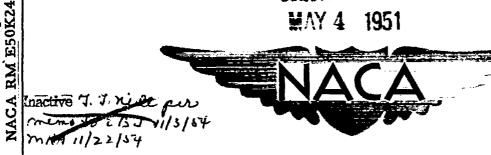
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# RESEARCH MEMORANDUM

COMPRESSOR PERFORMANCE CHARACTERISTICS OF A PYTHON

TURBINE-PROPELLER ENGINE INVESTIGATED IN

ALTITUDE WIND TUNNEL

By Emmert T. Jansen and John E. McAulay

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#### RESEARCH MEMORANDUM

COMPRESSOR PERFORMANCE CHARACTERISTICS OF A

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#### SUMMARY

An investigation has been conducted in the NACA Lewis altitude wind tunnel to determine the performance of the Python turbine-propeller engine. Compressor-performance data were obtained for a range of simulated altitudes at a single cowl-inlet ram pressure ratio while the engine was run over its full operable range of engine speeds. At each engine speed, data were obtained over a range of compressor pressure ratios by extracting various amounts of power from the turbine.

Over the range of conditions investigated at each altitude, the variation in compressor efficiency was small, which allowed the compressor a wide range of operation close to optimum efficiency. At constant corrected engine speed and turbine-inlet temperature, compressor efficiency decreased approximately 0.04 for an increase in altitude from 10,000 to 40,000 feet. The maximum compressor efficiency obtained was 0.804 and occurred at an altitude of 10,000 feet, a corrected air flow of 42.8 pounds per second, and a compressor pressure ratio of 3.8, which corresponds to a corrected engine speed of 7000 rpm and a turbine-outlet temperature of 1264° R.

The velocity profile at the compressor outlet was in general unaffected by changes in altitude, turbine-inlet temperature, or engine speed.

#### INTRODUCTION

An investigation to determine the static and dynamic performance characteristics at altitude of a Python turbine-propeller engine has been conducted in the NACA Lewis altitude wind tunnel.

As part of this investigation, performance data for the compressor operating as an integral component of the engine were obtained over a range of altitudes and engine speeds at a single cowl-inlet ram pressure ratio; at each engine speed, data were obtained over a range of compressor pressure ratios by extracting various amounts of power from the turbine.

The performance of the l4-stage axial-flow compressor is presented graphically to show the effects of variations in compressor pressure ratio and in engine speed and to show a general trend with altitude. Compressor performance deterioration caused by oil and dust accumulation on the compressor blades is discussed. A complete tabulation of the compressor performance data is also presented.

#### APPARATUS AND PROCEDURE

#### Engine

The Python turbine-propeller production engine has a static sealevel nominal rating of 3670 shaft horsepower and 1150 pounds of jet thrust at an engine speed of 8000 rpm and a turbine-outlet temperature of 590°C (1554°R). The maximum diameter of the engine is  $54\frac{1}{2}$  inches and the net dry weight is 3150 pounds. The main components of the engine include propeller-reduction gearing, a 14-stage axialflow compressor, 11 combustors spaced around the compressor casing, a two-stage turbine, a tail pipe, and a fixed-area exhaust nozzle.

#### Compressor

The compressor has an air-flow capactiy of approximately 54.2 pounds per second and a pressure ratio of 5.1 when the engine is operating at rated sea-level static conditions.

Air enters the engine through an inlet duct, which forms an annulus around the outer diameter of the engine (fig. 1) and is located approximately 60 inches aft of the rear propeller. A screen installed in this annulus prevents foreign objects from entering the compressor. From the inlet annulus, the air-flow passage is divided into 11 convergent throats and is turned inward

180° to the entry annulus of the compressor. The air passes forward through the compressor into the diffuser annulus, where it is turned outward 90° with the aid of guide vanes; the annulus is then divided into 11 combustion-chamber-inlet elbows, where the air is turned rearward 90° with the aid of guide vanes and passes into the combustion chambers.

To assist in starting and accelerating to an operable engine speed, air is bled from the compressor outlet to the atmosphere through a blow-off valve. During normal operation this blow-off valve remains closed. Air is bled from the fifth stage of the compressor to cool the rear bearing and the rear face of the turbine. Air is bled from the tenth stage to cool the front face of the turbine. Air leakage through labyrinth seals at the compressor outlet is piped back into the tail pipe.

Photographs of the compressor rotor and stator are presented in figures 2 and 3, respectively. The rotor blades of the first five stages are fastened on a uniformly tapered hub, whereas those of the final nine stages are fastened on a constant-diameter hub. The rotor-tip diameter is 20.35 inches through the first five stages and tapers to 17.75 inches at the fourteenth stage. The hub-tip ratios of the first, fifth, and fourteenth stages are about 0.61, 0.74, and 0.85, respectively.

#### Installation and Instrumentation

The engine was mounted in a wing section that spanned the 20-foot-diameter test section of the altitude wind tunnel (fig. 4). Refrigerated air was supplied to the engine from the tunnel air stream.

Pressures and temperatures were measured by instrumentation installed at several stations throughout the engine (fig. 1). Detail sketches showing the locations of instrumentation at stations 1, 2, 3, and 5 are presented in figures 5, 6, 7, and 8, respectively. Compressorstage static-pressure wall orifices were located in the planes of the leading edges of the rotor stages throughout the compressor.

#### Procedure

Performance data were obtained at an average cowl-inlet ram pressure ratio of 1.03 at pressure altitudes of 10,000, 20,000, 30,000 and 40,000 feet. Engine speed was varied from 6800 to 8000 rpm; at each engine speed, data were obtained over a range of compressor pressure ratio, which was changed by varying turbine-inlet temperature. The minimum engine flight idling speed was approximately 6500 rpm and was regulated by the engine control mechanism, which was preset at sea-level conditions. As altitude was increased to 40,000 feet, the minimum operable engine speed increased to approximately 7200 rpm. A power check run was made each day at a pressure altitude of 10,000 feet, cowl-inlet ram pressure ratio of 1.03, engine speed of 8000 rpm, and engine-inlet temperature of 59° F for four turbine temperatures. Preceding the power check run each day, kerosene was sprayed into the compressor inlet for 30 minutes while the engine was operating at idle speed in an attempt to clean the compressor-blade surfaces.

For all flight conditions except those of the power check runs, refrigerated air was supplied to the engine at the standard NACA temperature for each flight condition except that the minimum air temperature was about  $-25^{\circ}$  F.

The symbols and the methods of calculation are given in the appendix.

#### RESULTS AND DISCUSSION

The air-flow passage through the engine makes two 180° turns, one ahead and the other aft of the compressor rotor. The engine manufacturer furnished information that the energy loss caused by the turn ahead of the compressor rotor is 18 percent of the compressor-inlet velocity head (station 1) and that the energy loss resulting from the turn aft of the compressor is 34 percent of the compressor-outlet velocity head (station 2). Because of structural limitations of the engine, the compressor-inlet and compressor-outlet stations (stations 1 and 2) had to be so located as to include these energy losses in the pressure and temperature measurements. The inclusion of these losses in the compressor performance resulted in a reduction in compressor efficiency of less than 0.01 for all conditions investigated.

#### Deterioration of Compressor Performance

Oil leakage from the accessory gear box could enter the compressor inlet and provide a sticky surface on the compressor blades and turning vanes, which permitted an accumulation of foreign particles from the tunnel air stream on these surfaces. The use of kerosene sprayed into the compressor-inlet as a cleaning solution at the most retarded but did not eliminate the accumulation of foreign particles on the compressor surfaces. This accumulation of foreign particles resulted in a progressive decrease in compressor performance with increasing engine operational time. The decrease in compressor performance is shown in figure 9 for an engine operational time interval in the altitude wind tunnel of approximately 32 hours. At each altitude and engine speed as engine operation time increased, air flow and compressor pressure ratio decreased for a given turbineinlet temperature. The reduction in air flow required a decrease in engine fuel flow in order that the constant turbine-inlet temperature could be maintained. For this deterioration, however, the value of compressor efficiency remained unchanged. In order for the efficiency to remain constant, the actual work per pound of air of the compressor must decrease by the same ratio that the ideal work per pound of air decreases. An explanation of this characteristic is that in one or several of the leading stages of the compressor, the efficiency may decrease in such a manner as to improve the existing conditions of the air flow at some of the latter stages, with the over-all result that the compressor efficiency remained approximately constant. The loss in air flow and compressor pressure ratio with increasing engine operational time caused substantial reductions in shaft horsepower at a given turbine-inlet temperature.

The deterioration effect was determined only for the rated engine speed at the four altitudes investigated and the information is insufficient to permit adjusting the data for all engine speeds. Because of this deterioration, quantitative evaluation of variations in performance cannot be made for various altitudes. For any given altitude except 10,000 feet, however, the performance data were obtained over an engine operational time interval of  $3\frac{1}{2}$  hours or less. Performance data for the rated engine speed at an altitude of 10,000 feet were obtained 10 hours earlier than the performance data for the other engine speeds at this altitude. The maximum change in compressor characteristics for the time interval of  $3\frac{1}{2}$  hours is a 1-percent decrease in corrected air flow and in compressor pressure ratio while the compressor efficiency remained

approximately constant. Quantitative evaluation of variations in performance can therefore be individually made for each altitude.

#### Compressor Performance

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Compressor efficiency. - Compressor efficiency is presented in figure 10 as a function of corrected turbine-inlet temperature for the four altitudes investigated. The turbine-inlet temperature is used as the independent variable because at constant engine speed with a choked turbine nozzle the temperature is proportional to the square of the compressor-outlet total pressure and thus provides a connecting link for the compressor performance with engine and turbine performance. In general, the change in compressor efficiency was relatively small over the range of conditions investigated at each altitude, which allows the compressor a wide range of operation near optimum efficiency. For all altitudes and flight conditions investigated, the compressor efficiency variation was only from 0.804 to 0.695. The maximum compressor efficiency of 0.804 was obtained at an altitude of 10,000 feet, a corrected air flow of 42.8 pounds per second, and a compressor pressure ratio of 3.8, which corresponds to a corrected engine speed of 7000 rpm and a turbine-outlet temperature of 1264° R. At each altitude, the decrease in compressor efficiency that accompanied a change from minimum to maximum engine speed at constant corrected turbine-inlet temperature varied between 0.03 and 0.06; at any given engine speed the trend was for an efficiency rise of from 0.01 to 0.04 as turbine-inlet temperature was varied from minimum to the maximum allowable value.

The data in figure 10 are cross-plotted in figure 11 to illustrate the effect of altitude on compressor efficiency at constant corrected engine speed and turbine-inlet temperature. At constant values of corrected engine speed and turbine-inlet temperature, compressor efficiency decreased a maximum of only 0.04 for an altitude increase from 10,000 to 40,000 feet. Because compressor performance deterioration did not affect compressor efficiency, the loss in efficiency with increasing altitude may be largely attributed to the Reynolds number effect on the compressor performance.

Compressor performance maps. - Compressor performance maps with contours of constant compressor efficiency and lines of constant corrected turbine-inlet temperature superimposed are presented in figure 12 for altitudes of 10,000, 20,000, 30,000, and 40,000 feet. If the corrected turbine-inlet temperature is decreased at constant corrected engine speed, the corrected air flow increases, with the

increase in air flow being greater at low engine speeds. A decrease in corrected engine speed at constant corrected turbine-inlet temperature results in decreases in corrected air flow and in compressor pressure ratio. Because the altitude effect and compressor deterioration effect are inseparable, it is not possible to accurately evaluate the shift in lines of constant corrected engine speed and contours of constant compressor efficiency with changes in altitude. At constant corrected engine speed and turbine-inlet temperature the general trend of the compressor performance with increase in altitude is a decrease in air flow and in compressor pressure ratio.

#### Velocity and Static-Pressure Profiles

The velocity profiles at the compressor outlet are shown in figure 13. Figure 13(a) represents the compressor-outlet velocity profile for a single engine and flight condition. Individual radial pressure measurements for rakes equally spaced circumferentially from a radial center line through the compressor-outlet passage have been averaged. Figures 13(b) to 13(d) represent the velocity profiles at the compressor outlet for various operating conditions. In every case the velocity is lower at the inner wall, which may be caused by the elbow preceding this station. The data indicated no general effect on the velocity profile with variations in altitude, corrected turbine-inlet temperature, or corrected engine speed.

The rotor-stage static-pressure ratios for ranges of altitude, corrected turbine-inlet temperature, and corrected engine speed are presented in figure 14.

A complete tabulation of compressor performance data is presented in table I and compressor performance deterioration data in table II.

### SUMMARY OF RESULTS

From an investigation of a Python turbine-propeller engine in the NACA Lewis altitude wind tunnel over a range of simulated altitudes and at a cowl-inlet ram pressure ratio of 1.03, the following results relating to the compressor were obtained:

1. The variation in compressor efficiency was small for all operating conditions investigated at each altitude, thereby permitting the compressor a wide range of operation close to optimum efficiency.

- 2. Increasing altitude from 10,000 to 40,000 feet at a given corrected engine speed decreased compressor efficiency approximately 0.04 for all corrected turbine-inlet temperatures. This loss in compressor efficiency with increasing altitude may be largely attributed to the Reynolds number effect on the compressor performance.
- 3. The maximum compressor efficiency of 0.804 was obtained at an altitude of 10,000 feet, a corrected air flow of 42.8 pounds per second, and a compressor pressure ratio of 3.8, which corresponds to a corrected engine speed of 7000 rpm and a turbine-outlet temperature of 1264° R.
- 4. In general, the velocity profiles at the compressor outlet were unaffected by changes in altitude, turbine-inlet temperature, and engine speed.

Lewis Flight Propulsion Laboratory,
National Advisory Committee for Aeronautics,
Cleveland, Ohio.

# APPENDIX - CALCULATIONS

## Symbols

The following symbols are used in the calculations:

<b>A</b> .	area, sq ft
a	stagnation speed of sound in air, ft/sec.
c <sub>p</sub>	specific heat at constant pressure, Btu/(lb)(OR)
CT	thermal expansion ratio, ratio of hot exhaust-nozzle area to cold exhaust-nozzle area
D.	compressor rotor-blade tip-diameter, ft
ghp	gear horsepower
g	acceleration due to gravity, 32.2 ft/sec2
н .	enthalpy, Btu/lb
М	Mach number
N	engine speed, rpm
P	total pressure, lb/sq ft absolute
р	static pressure, lb/sq ft absolute
R .	gas constant, 53.4 ft-lb/(lb)(OR)
shp	shaft horsepower
T	total temperature, <sup>O</sup> R
T <sub>i</sub>	indicated temperature, OR
t	static temperature, OR
υ	compressor rotor tip speed, ft/sec
v ·	velocity, ft/sec
Wa	āir flow, lb/sec
W <sub>f</sub>	fuel flow, lb/hr

 $W_{\mathbf{g}}$ 

 $W_{\mathbf{c}}$ 

 $W_{rb}$ 

 $W_{t}$ 

gas flow, lb/sec

compressor leakage air flow, lb/sec

turbine cooling air flow, lb/sec

rear bearing cooling air flow, lb/sec

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r	ratio of specific heats
δ <sub>1</sub>	ratio of absolute total pressure at cowl inlet to absolute static pressure at NACA standard atmospheric sea-level conditions
$\theta_{1}$	ratio of absolute total temperature at cowl inlet to absolute static temperature at NACA standard atmospheric sea-level conditions
$\eta_c$	adiabatic compressor efficiency
Subscripts	:
c	compressor
t	turbine
0	free-stream conditions
1	cowl or compressor inlet
la	compressor rotor stages
2	compressor outlet
3	turbine inlet
4	turbine outlet
5	tail pipe
Generalizi	ng parameters:
$N/\sqrt{\theta_1}$	corrected engine speed, rpm
$T_3/\theta_1$	corrected turbine-inlet total temperature, OR
$W_{a,1}\sqrt{\theta_1}/\delta$	corrected engine-inlet air flow, lb/sec

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#### Methods of Calculation

Temperatures. - Static temperatures were determined from indicated temperatures with the following relation:

$$t = \frac{T_1}{1 + 0.85 \left[ \left( \frac{P}{p} \right)^{\frac{\gamma - 1}{\gamma}} - 1 \right]}$$
 (1)

where 0.85 is the impact recovery factor for the type of thermocouple used.

Temperature measurements were obtained by means of thermocouples at all stations except the turbine inlet, station 3. The high temperature level and difficulty of installing instrumentation at this station would not permit the measurement of temperature with thermocouples. The turbine-inlet total temperature was therefore evaluated in the following manner: The work of the turbine is equal to the work of the compressor plus the work available for the propeller along with the gear losses. In terms of total enthalpy the relation is

$$W_{g,3}(H_3 - H_4) = \frac{550}{778} (shp + ghp) + (W_{a,2} H_2 - W_{a,1} H_1)$$
 (2)

where

$$W_{a,2} = W_{g,3} - \frac{W_{f}}{3600}$$

and

$$H_3 = c_p T_3$$

The shaft horsepower was obtained from a torquemeter reading. The gear horsepower was obtained from a curve supplied by the engine manufacturer showing gear horsepower as a function of shaft horsepower.

Gas flow. - Gas flow through the tail pipe of the engine may be determined using pressure and temperature measurements at station 5 by the equation

$$W_{g,5} = p_5 C_T A_5 \sqrt{\frac{2\gamma_5 g}{(\gamma_5 - 1)Rt_5} \left[ \frac{p_5}{p_5} \right]^{\frac{\gamma_5 - 1}{\gamma_5}} - 1}$$
 (3)

where  $C_{\mathrm{T}}$  is the correction for thermal expansion of the exhaust nozzle. The turbine gas flow is

$$W_{g,3} = W_{g,5} - W_{c} - W_{t}$$

where compressor leakage and all cooling air flow were determined from pressure and temperature measurements. This calculation of gas flow gave values having the correct magnitude but the scatter was excessive owing to the difficulty in measuring the small dynamic pressures.

Because the turbine nozzle was choked for the range of conditions investigated allowing the assumption to be made that the turbine - nozzle vena contracta area is constant, the following equation was used to obtain the final calculated gas flow:

$$W_{g,3} = \sqrt{\frac{g}{R}} \frac{P_3}{\sqrt{T_3}} \frac{A_{3,av}\sqrt{\gamma_3}}{\frac{\gamma_3+1}{2(\gamma_3-1)}}$$

$$\left(\frac{1+\gamma_3}{2}\right)^{\frac{2(\gamma_3-1)}{2}}$$

in which the average turbine-nozzle vena contracts area was calculated from equation (4) using the tail-pipe (station 5) gas flows and turbine-inlet total temperature based on tail-pipe gas flow. Using this average effective turbine-throat area and turbine-inlet temperature, the turbine gas flow was determined from equation (4). With this turbine gas flow, a recalculation was made for turbine-inlet temperature, which showed a negligible change in the recalculated temperature from the original calculated temperature. The error in turbine gas flow is the square root of the ratio of the two temperatures and therefore can be neglected.

Air flow. - Engine-inlet air flow is

$$W_{a,1} = W_{g,3} + W_{rb} + W_c + W_t - \frac{W_f}{3600}$$
 (5)

which is the air flow used throughout this report.

Compressor efficiency. - Adiabatic compressor efficiency was calculated using the following equation:

$$\eta_{c} = \frac{\left(\frac{P_{Z}}{P_{1}}\right)^{\frac{\gamma-1}{\gamma}} - 1}{\left(\frac{T_{Z}}{T_{1}} - 1\right)} \tag{6}$$

where  $\gamma$  is based on the average temperature of the air in the compressor.

Compressor Mach number. - The compressor Mach number is defined as the ratio of the tip speed of the compressor first-stage rotor blade to the speed of sound in air at the total temperature of the engine inlet air. The equation used is

$$M_{\rm C} = \frac{U}{a_1} = \frac{\pi DN}{60\sqrt{\gamma gRT_1}} \tag{7}$$

Compressor-outlet velocity. - The compressor-outlet velocity was determined by the equation

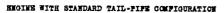
$$V_2 = \sqrt{\frac{2\gamma}{\gamma - 1}} gRT_2 \left[ 1 - \left(\frac{p_2}{P_2}\right)^{\frac{\gamma - 1}{\gamma}} \right]$$

where individual total pressures and average static pressures and total temperatures were used.

TABLE I - PERFORMANCE DATA OF PYTHON TURBINE-PROPELLER

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Outlot a vane sta (1b/aq f	Outlet Ature,	Outlet tot pressure (lb/sq ft	Outlet pressur (lb/sq	Turbine- pressur (1b/sq 1	Total te Ts (oR)	Total p	tatio Lb/eq	mprest t10,	Correcte speed, K	Compress number,	Tail-pi	furbine-	Cowl-inlet Wa,l (1b/	Wa,1 /0	Turbine-ini temperature (on)	Compressor ency, ho	Average time in tumel (	Run
7255	869	8063	7910	7762	1452	1636	1478	5.404	8278	0.659			39.68	54.42		77.4	3°5 4 11 12	ř.
7090	868 863	7905 7826	7755 7674	7608 7533	1389 1352	1630 1625	1478	5.291	8278 8302	.659 .661	39.97 39.84	39.27 39.32 39.60	39.71	54.58 54.57	2094 2016 1965	76.4	3.2	3
6560	855 851	7417	7260 6777	7150 6662	1206	1602 1584	1465 1467	4.985	8286 8334		40.65	30.03	40.10	EK.00	1768	75.2	3.2	4
6871	860	7627	7492	7341	1486	1620	11476	5.115	8040	.640	41.03 37.51	41.00 36.90 37.07	87.30	51.39	1487 2108	74.5 77.5	3.2 13.1	5
6707 6602	851 850	7465 7373	7329 7234	7184 7095	1352	1597 1603	1458	5.061 4.952	8054 8056	.642						77.4	13.1	8
6230 5571	835 832	7054 6488	6907 6324	6779 6213	1186	1583	1462	4.744	8079	-643	38.51	38.37 38.40 35.77 36.09 36.25	38.74	53.26	1713	76.7	13.1	9
6523	835	7240	7111	6969	1014	1555 1602	1456	4.369	8032 7865	.626	57.68 36.37	55.77	36.18	49.76	1443 2041	74.4	13.1	끪
6467 6336	832 833	7199	7066 6947	5925 5815	1391	1598 1591	1469	4.845	7878 7849	.627 .625	36.59 35.95	36.09	36.50	50.82	1987	78.5	15.9 15.9	18
5925	822	6713	6572	6450	1160	1570	1460	4.521	7857	.625	01.00	01.00	01.32	0T.05	TOOT	77.2	13.9	14
5355 6132	806 821	6250 6809	6088 6688	5985 6557	960 1416	1543 1585	1451	4.226	7895 7643	.628 .608	37.19 34.07	38.06 34.12	38.42 34.52	52.97 47.53	1390 1982	75.3	13.9	15
6054 5949	817 820	6738 6652	6615 6527	6484 6399	1374	1575	1462		7643	.60B	34.06	34.22	34.62	47.92	1932	79.2	14.6	17
5577	816	6317	6185	6072	1313 1166	1562 1564	1472 1459	4.251	7621 7599		35.06	34.57 35.04	35.41	48.12 49.15	1840	78.5 77.5	14.6	18
5146 5746	797	5996 6392	5845 6278	5742 6155	978 1358	1546 1567	1460 1467	4.032	7643 7414	.608 .590	35.81 32.45	36.36 32.83	36.71	50.62	1363	78.5	14.6	20 21
5683	805	6339	6222	6101	1322	1565	1465	4.257	7414	.590	32.93	35.34	33.75	46.58	1829	79.1	14.8	22
5603 5341	799	6279 6058	6158 5928	6043 5818	1249 1134	1559 1557	1460 1459	4.228	7443 7456	.592	35.53 35.25	33.74 34.16	54.14 54.53	47.10	1752	78.7 78.3	14.8	23
4881 5068	788 773	5671 5645	5530 5543	5435 5437	989 1298	1526	1448	5.845	7407	-590	55.59	34.40	34.78	48.52	1377	77.1	14.8	25
5075	778	5671	5565	5474	1284	1541 1550	1456	3.798	7002 5996	.557 .557	30.71 31.38	30.12 30.76	30.41	48.12 49.15 50.62 45.88 46.58 47.10 47.55 48.52 42.08 42.90	1786 1715	79.9	17.6 17.6	27
4919 4803	766 764	5531 5468	5425 5352	5325 5254	1192 1106	1546 1535	1457	5.712 5.662	7002	.557 .558	32.79 31.28	30.82 31.64		42.92 43.90	1626 1515	80.2 79.3	17.6 17.6	28
4504	754	5225	5097	5007	880	1527	1455	3.512	7023	.559	32.84		32.22	44.43	1366	78.3	17.6	30
5125	844 839	5673 5620	5575 5532	5455 5407	1559 1461	1101	983 989	5.715	8542 8542		27.14 27.58	26.81	27.10	54.12	2544	75.2	19.6	31
4953	834	5531	5442	5316	1389	1102	991	5.520	8550	.681	27.75	27.47	27.74	54.54 54.86	2246	75.5 75.1	19.6	32 33
4544	827 812	5222 4788	5119 4678	5017 4587	1252	1079	980 977	51254 4.798	8550 8558	.681		27.41 27.74	27.67	55.16 55.47	1943	73.7 71.3	19.6	34 35
4952	827	5460	5368	5251	1479	1095	986	5.482	8329	.663	26.75	26.32	26.59	52.93	2254	76.4	20.2	36
4861 4735	821 819	5398 5280	5303 5184	5189 5080	1345	1098 1092	981	5.362	8321 8321	.662 .662		26.63 26.78	27.04	53.24 53.57	2167	76.9 76.3	20.2	37 38
4730 4428	813 807	5277	5179 4906	5075 4808	1309 1175	1086	985	5.288	8360 8345	.665	27.45	27.10	27.36	54.16	2037	76.1	20.2	39
3986	793	4623	4513	4431	989	1064	977	4.623	8360	.665	20 37	27.22 27.57	27 70	E4 00	1826 1547	74.9 72.4	20.2	40
4679 4598	809 812	5176 5105	5088 5015	4985 4915	1427 1584	1084	984 993	5.080	8116 8070	.646 .642	26.06 26.07	25.56	25.82	51.40	2177	77.2 77.0	20.9	42
4595	804	5106	5016	4913	1362	1078	983	5.132	8116	.646	25.94	25.78	26.02	52.16	2085	77.5	20.9	44
4532 4242	802 793 .	5058 4800	4969 4704	4870 4512	1305 1172	1082 1057	989	5.043 4.839	8108 8123		26.21 25.84	25.56 25.61 25.78 26.11 26.22	26.35	52.16 52.83	1997 1802	77.3 76.1	20.9	45
3846 4428	781 789	4464 4887	4359 4805	4280 4704	979 1405	1051	971 984	4.491	8123 7917	.647	E1.00	40.00	A W	04.02	1510	74.3	20.9	47
4351	785	4825	4741	4648	1564	1064	977	4.874	7932	.631	24.65	24.50	24.73	49.34	2131 2083	77.5 77.1	21.5 21.5	48 49
4325 4038	780 776	4811 4555	4727 4462	4633 4378	1288 1167	1068	983		7954 7917		25.08	25.09	25,32	50.05	1984 1786	76.9	21.5 21.5	50
3681	761	4251	4169	4095	961	1044	978	4.264	7932	.631	26.30	26.07	26.27	52.05	1477	76.4 74.8	21.5	51 52
4144	772 773	4591 4519	4514	4423 4353	1377 1392	1059	980		7702 7688	.613	25.60 25.00	25.55	23.54	46.88	2070	77.7 77.4	22.0	53 54
4056	775	4502	4417	4330	1364	1052	975		7666	510	23.27	22,98	23, 19	46.58	2028	77.7	22.0	55



TABLE I	-	PERFORMANCE	DATA	OF	PYTHON	TURBINE-	-PROPELLER
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[			Γ	1	_					1		<del></del> :							·· <del>-</del> · ·			7		
		1 .		J.			Cow	1 1n	$\overline{}$	1									-					
		1 Od		1.			•	2	4	Į.														Í
		25	*	104,	2	•	1	-	ē-	1	c.	mpres	80P-7	otor-	atage	stat	ic pr	osaur	e. (1	.b/so	ft al	os.)		
1 1		F 7	å,	티	horsepowe	H 20	erat	abs.	aba.		•	ALL U		0.01			,.			,				1
		# 🔅	2	fuel	2	10.4	di di	0 4	1	]									•					
1 1	eg eg	발표 기계			å		472	툂							4			<u> </u>						
] _ [	# C	, ,	A P	à à	3	0 7 1	7		35							Btag	8						E	,
E E	Alts (ft)	Coar	Engine (rpm)	<b>188</b>	Shad of		Total		(Lb)	1,	8	5	4,	5	6	. 7	8	9	10	11	12	13	14	
58 57	20,000	1.026	7205	1225	1246 1134	963 971	458 455	988	966	837 832	971 966	1119	1274	1429	1619 1600	1837	2083 2064	2344 2332	2625 2599	2935 2916	===	3583 3585	3872 3881	ļ
58	1	1.026		925 645	773	972	456	997	978	813	954	1095	1243	1390	1575	1778	2017	2263	2531	2827		5439	3700	ĺ
59		1.025	6805	940	249 816	985	454	1989		788	929	1070	1203	1337 1398		1696	1914	2140 2201	2586 2435	2701 2675		3817 3236	3428	ı
61	` '	1.028	1000	875	720	967	454	994	973	851	985	1118	1253	1386	1565	1738	1957	2169	2414	2629		5205	3407	ĺ
62 63		1.025		900 595	582 255		455	990		848	969		1243	1376 1356		1714		2144	2369 2321	2574 2259		5136 3046	3355 3201	ŀ
64	50,000	1.025	8006	1265	1576	626	437	645		472	528	641	754	873	1035	1218			1944	2239		2901	5285	t
65		1.027	1	1170	1303	626	488	643		465		634	747	866		1197			1901	2204		2888		i
66		1.027		1085	1173 669		487 436	645		467	525	629 608	735 714	854 819	1002 960	1171 1115		1615 1530	1861 1755	2150		2783 2643		!
68		1.026	ŧ	575	266	621	436	637	617	450	498	584	682	781	894	1027	1203	1400	1598	1844		2565	2604	
70	•	1.026	7806	1160	1287 1215		458 458	645	623			669 648	782 761	902 660	1056	1232	1457	1669°	1901	2162		2795 2774	3084	į
71		1.027	ĺ .	983	1077	625	487	642	623		536	655	754	875	1,014	1190	1880	1806	1858	8118		2711	2992	
72		1.027	}	815	773	625	487	640		471	527	655	752	844	271	1126	1816	1597	1745	2013			2857	l
78		1.027	7606.	570 1050	336 1173	619 624	436 439	636	616 623		569	624	716 794	614 914	934 1054	1068 1230	1257 1428	1427 1639	1624 1864	1856 2125		2565 2695	255	ĺ
75		1.024	]	1020	1114	630	488	645	627	496	574	679	792	905	1045	1214	1419	1650	1848	2123		2695		1
76	j	1.025	1	774	1016		444	639		495	575	678 660	784 759	897	1037	1192	1389 1322	1586 1512	1805 1723	2058 1956		2593 2470		
78		1.026		540	278	618	457	634				637	722	820		1067	1236	1412	1602	1820		2352	2489	-
79		1.025	7406	960	1052		437	644				. 706	805	918		1820	1403		1012			2566	2811	ĺ
80		1.027	İ	910 845	995	626 625	437	643	624	515	586 576	705 690	804 789	917	1028	1198		1871	1776 1760	2008		2522 2495	2732	
82		1.027		720	664	626	488	643	625	502	580	685	777	882	1002	1157	1386	1509	1699	1931		2403	2621	l
85 84		1.027	7205	525 840	302	623 626	487	640	622 627		868	663 913	755	853 917	966 1045	1100 1184		1431 1543	1614	1825 1952		2275 2425	2641	•
85		1.026	1208	805		626	435	642	628			705	811	910	1043		1367	1543	1740	1959		2430	2541	ĺ
86		1.027	}	765		625	455	842		515		898	797	905		1110			1712	1930		2388	5609	1
87 88		1.027		665 500	573 288		456 456	640		507	585 579	690 684	775 789	880 860	973	1141	1303 1255		1662 1665	1875		2517 2184	2514 2553	Í
89	}	1.027	8805	670	580	683	441	640	627	550	620	719	803	895	1000	1127	1275	1430	1592	1768		2141	2317	1
90		1.026		658	551	624	436	640	627			719	803 ELO	888 887	1000	1127	1275	1423	1584 1577	1760 1760		2184	2503 2503	1
91 92		1.027		595 525	481 565	630 624	440	647 641	653 627	550 558	654 615	706	791	875	988	1101	1242	1382	1544	1720		2079	2827	
95		1.025	1	472	262	629	439	645	651	543	620	704	789	875	979	1092	1232	1380	1828	1897		2049	2190	١.
94		1.025	8006	755		396 396	441	406	394	291	528 527	391 411	475 482	553 552	644 645	757	891 883	1039	1194 1178	1370 1362		1771 1742	1868	1
96		1.028	ì	660	671	392	441	403	391	288	331	401	464	535	626	725	852	999	1147	1323		1703	1,894	1
97		1.026	i	560	507	390	439	400	569 567			587 580	457 436	521 493	598 570	697 647	909 753	943 873	1084	1253		1612 1492	1788	ĺ
98	,	1.025	7806	700	252 745	389 397	446 434	399 407	397	508		399	484	561	652	758	885	1026	1175	1549		1708	1899	(
100		1.025	1	670	699	394	456	404	398	505	833	596	481	551	642	748	875	1009	1149	1518		1684	1860	ŀ
101		1.020	]	650 400	632 237	391 395	457 458	399 404	388 393			385 399	485	541 512	625 596	731 674	851 772	984 892	1019	1294 1166		1660 1490	1645	1
103		1.028	7806	620	626	391	458	402	391	316	351	414	492	562	639	745	885	991	1125	1280		1618	1787	į
104		1.030	1	590	593	394	438	406	395			418 401	489	552	656	735 711	848	974 950	1108 1077	1270 1239		1801 1563	1765 1726	ł
105		1.028	1	550 490		391	442	408	391 397	503	358 552	415	486	535 542	619 626	711	825	945	1077	1225		1549	1703	l
107	[	1.026	-[	400	255	<b>\$90</b>	439	400	389	295	558	401	464	521	591	678	774	887	1007	1147		1450	1891	
108		1.028	7406	520 470	491	594 595	442	405		313	369 362	432 418	496 482	559 545	636 622	721	878	946 918	1066	1800 1178		1502	1645	į
109		1.028	ŀ	390		595	444	405	394		377	426	483	539	616	687	785	891	1003	1157		1418	1846	ĺ



ENGINE WITH STANDARD TAIL-PIPE CONFIGURATION - CONCLUDED

						_											_	
	Compr	essor			,	fail ;	ipe		· · · ·					Corre	cted		, I	
Outlet straightening- ware static pressure (1b/sq ft abs.)	Outlet total temper- ature, Tg (OR)	Outlet total pressure, Pg (1b/sq ft abs.)	Outlet statio pressure, pg (lb/sq ft abs.)	Turbine-inlet total pressure, Ps (1b/sq ft abs.)	Total temperature, T <sub>5</sub> ( <sup>o</sup> R)	Total pressure, P5 (1b/sq ft abs.)	Static pressure, ps (1b/sq ft abs.)	Compressor pressure ratio, Pg/Pl	Corrected engine speeds N//61 (rpm)	Compressor Mach number, Mo	Tail-pipe air flow, Wa,5 (lb/sec)	Turbine-inlet air flow, Wa,5 (lb/sec)	Cowl-inlet air flow, Wa,l (lb/aec)	Wa, 1 /61/51 (15/mes)	Furbine-inlet total temperature, 73/9	Compressor effici- ency, 4 <sub>G</sub> (percent)	Average engine running time in altitude wind tunnel (hr)	Run
4048 4050 5812 3477 5580 5580 5583 3201 3283 3283 3283 3283 3283 3283 3283 328	775 7699 7644 7653 7366 7326 831 827 823 831 844 799 800 795 800 795 767 774 772 774 775 775 775 775 775 775 775 775 775	4482 4515 44515 44515 44515 3986 5928 5708 5715 5617 3617 3617 3617 3617 3617 3617 3617 3	4408 4435 4435 4230 3959 3916 3852 3767 3625 3763 3586 3586 3586 3586 3586 3586 3586 35	4518 4546 4149 3865 3791 3557 3557 3574 3508 3087 32418 2357 32418 2358 3359 32418 2858 3359 32418 2858 3359 32418 2858 2858 2858 2865 2865 2865 2865 286	1356 1296 1145 973 1159 1159 1159 1159 1208 1159 1262 1262 1340 127 121 121 1340 121 121 1340 121 1340 121 1340 121 1340 121 1340 121 1340 121 1340 121 1340 121 1340 121 1340 121 121 1340 121 1340 121 1340 121 121 121 121 121 121 121 121 121 12	1050 1058 1047 1039 1032 1031 1026 718 718 719 703 694 770 770 770 770 676 677 678 687 687 687 687 688 684 675 688 688 688 688 688 688 688 688 688 68	974 9819 979 970 973 980 975 975 638 636 638 636 638 631 632 632 632 632 632 632 632 632 632 632	4.536 4.527 4.331 4.078 3.986 3.985 3.727 5.883 3.727 5.429 4.928 5.603 5.429 5.429 5.429 5.429 5.429 5.429 5.429 5.429 5.429 5.429 5.429 5.429 5.429 5.429 5.429 5.429 6.429	7886 7695 7688 77092 77688 77268 77268 77268 77268 8727 8719 8755 8755 8755 8755 8755 8755 8755 875	0.610.616.616.616.616.616.616.616.616.61	25.24 25.85 24.50 22.85 25.35 17.80 17.80 17.80 17.82 17.72 17.72 17.72 17.72 17.73 17.85 17.85 17.85 17.85 17.85 17.85 17.85 17.85 17.85 11.88 17.85 11.88 17.85 11.88 11.88 11.88 11.88 11.88 11.88 11.88 11.88 11.88 11.88 11.88 11.88 11.89	25.01 28.72 24.61 24.61 22.06 22.06 22.16 23.98 17.85 18.95 18.95 10.99 11.95 11.95 11.95 11.95	25.22 23.95 24.35 24.35 22.35 22.35 25.16 18.02 18.01 18.08 18.33 18.12 17.46 17.90 17.40 17.90 17.40 17.59 16.12 17.39 16.12 17.39 16.52 16.68 16.68 16.72 16.55 16.68 16.72 11.10	46.75 47.55 49.63 49.63 44.88 46.04 54.42 74.58 46.04 55.21 55.21 55.21 55.22 55.23 55.23 55.23 55.23 55.23 55.23 55.23 55.24 49.80 55.24 49.80 55.24 49.80 55.24 49.80 55.24 55.25 55.35 55.36	2016 1952 1470 1728 1470 1728 1430 1632 1430 22468 2576 2468 2400 2299 2214 1700 2278 2216 1622 2214 2214 2214 2214 2214 2214	77.5 77.7 77.7 78.0 78.0 78.0 78.0 78.0 78.0	22.000.05.5.5.5.2.2.2.2.2.2.2.2.2.2.2.2.	56 57 58 59 60 61 62 65 65 66 67 76 77 77 80 81 82 83 84 85 86 87 77 77 88 81 82 83 84 86 86 86 86 86 86 86 86 86 86 86 86 86
1732 2011 1973 1942 1760 1871 1861 1916 1788 1554 1728 1678 1678	813 814 812 809 797 802 799 800 791 783 776 773 772	1995 2233 2196 2171 1975 2095 2077 2036 2019 1901 1934 1889 1836	1952 2195 2155 2155 2156 1934 2061 2042 2000 1983 1862 1900 1861 1801	1908 2152 2115 2087 1890 2013 1998 1956 1957 1820 1857 1816 1763	1181 1470 1425 1365 1120 1415 1361 1308 1243 1151 1310 1251 1122	425 446 441 435 430 438 438 423 431 430 424	391 402 399 395 397 395 395 401 392 597 396	5.000 5.486 5.456 5.441 4.889 5.211 5.116 5.065 4.949 4.753 4.775 4.676	8638 8540 8516 8509 8501 8283 8283 8245 8245 8245 8268 8028 8006	.688 .690 .678 .677 .677 .659 .658 .658	10.85 10.83 10.72 10.80 10.73 10.45 10.74 10.47 10.85 10.89 10.07	11.08 10.81 10.77 10.89 11.03 10.38 10.48 10.48 10.68 10.57 10.02	11.17 10.98 10.88 10.98 11.13 10.48 10.59 10.58 10.78	54.90 51.94 52.24 53.53 50.66 50.68 51.38 51.38 51.93 48.63 49.05	1768 8363 8290 2197 1792 2238 2174 8071 1974 1795 2046 1957	70.6 71.1 71.7 72.6 69.5 72.0 71.6 72.3 71.1 74.1 73.6 72.6	31.6 50.4 50.4 50.4 52.0 52.0 52.0 52.0 52.0 52.0 52.8 58.8	98 99 100 101 102 103 104 105 106 107 108 109 110





TABLE II - PERFORMANCE DETERIORATION DATA FOR FYTHON

														PP 11											
		" pres-	H	flow, Wg	Wer,	-	•	r imi	E. 77		Com	press	or-r	otor-	stage	stati	o pre	ssure	, (11	)/sq 1	t aba	)			202
	epart:	fulet ran ratio, P <sub>1</sub> /	, peeds ec	fuel	t horsepow	Tunnel statio pressure, po (1b.sq ft abs.)	L tempera	Total pressur (lb/sq ft abs.	11				· · · · ·	·			-			,	<del></del>				· ·
2	(Ft.)	Com!	Engin	Inglas (1b/hr	Shaft	D. C	9 -	8.2	Statio (1b/eq	<u> </u>	-	3		-	-	Stag	68	-	130	11	12	1,5	174		
-			Д.		1 47 47	100	-	Aver		ngine	Mini		line:	8.0	HFB.			1 -	10	1 24	12	1.5	14		
1	10,000	1.087	8006	8580	2589	1486	618	_							2388	9652	3081	3539	4038	4644	5228	5968	6606		
3	,	1.027		2100	2246 1851	1455	518	1492	1447	1140	1316	1549	1760	1992	2261 2253	2591	5006	3435	3914	4477	5047	5759	6378		
4		1.026		1560	1328	1460	522	1498	1454	1137	1250	1538	1748	1961	2221	2510	2876	3284	3693	4228	4748	5398	5924		5. A##
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5 6	10,000	1.027	8006	8100	2510 2195	1447	514	1498		1149	1332	1572	1763	2008	2549 2290	2593	2894	3425	13881	4485	8007	5718	6324	·	-
7 8		1.027		1870	1831	1455	525	1494	1449	1151	1334	1557	1771	11996	2264 2197	2559	2939	3355	3798	4347	4883	8572	6250		
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10	10,000	1.025			2488 2118										2592 2528								6418		
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13		1.026		1100	349	1458	517			ngine					2196	2407	FROE	oros	3838	14019		4448	3446	<del></del>	
14	10,000	1.026	8006		2039			11488	1448	1181	1599	1.624	11856	12103	2408	2757	3166	2605	4053	4574	]	5665	6165	· ·	
15	-	1.028	l	1810	1376	1456		1484	1447	1175	1351 1351	1598	1795	2070	2359	2590	2971	3485 3551	3921 3746	4210		5153	5498	1	
							<u> </u>	AV61	ago i	ngine	run	ning	time,	4.5	hrs.		·								
17	20,000	1.028	8005	1730	1922	975	451	1009	970 965	728	812		1129		1498	1734								İ	
19		1.028		1475	1556	969	450	996	964	723	800	955	1105	1251	1448	1666	1955	2978	2630	3067	3503	4067	4566	1	
20		1.027	1	1265	1146	969		995	962 963	725 715	786 769	917	1082		1406 1353		1769	2043	8225	2712	3085	3585	5986	ĺ	_
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88	30,000	1.029	8006	1230		625	441	648	622 627	471	527 526		738 724	844 829	978 963	1125	1544	1583	1630	2096	2500	2823	3175		
24		1.027		1045	1107	624	441	641	621	487	526 506	605	711	802	929	1077	1274	1492	1731	5050	2316	2689	3034		
25		1.029	Ì	870 610	824 506	625 629	443	641	626	457 465	506 495		696 676	781 768	900 866					1914					
										ngine			time,	10.1	hrs.			<del></del>			<del></del>	<del></del>	7 2.1		
27	40,000	1.026	8006	745	768 746	390 387		400 597	388 385	285	321 553		454 452	511 516	600	694 692	820 81.B			1290			1926		
29		1.028		865	678	386	439	397	584 586	289	325	368	444	508	585	677	798	937	1078	1254	1444	1662	1886		
30		1.028		570 450	526	387 396	440	598 407	386 398	291	327 525		439	495	873 862	658	763 738	897 865		1200 1155			1784 1695		
			Ь.								<u> </u>	-	-	****		1				-				· <del></del>	





	ompre	ssor			Ta	11 pi	P. B.	1			1	İ		Corre	cted		1 1
straighte tatio pres ft abs.)	Outlet total temper- ature, T2 (OR)	Outlet total pressure, Pg (lb/sq ft abs.)	Outlet statio pressure, pg (lb/sq.ft abs.)	Turbine-inlet total pressure, P5 (1b/sq ft abs.)	Total temperature, To (OR)	Total pressure, P5 (1b/sq ft abs.)	Static pressure, pg (lb/sq ft abs.)	Compressor pressure ratio, Pg/Pl	Corrected engine speed, W//Gl (rps)	Compressor Mach number, Mo	Wash (1b/sec)	Turbine-inlet air flow, Wa,3 (lb/sec)	Cowl-inlet air flow, Ma,1 (lb/sec)	Cowl-injot air flow, Ma,1 /61/61 (lb/sec)	Turbine-inlet total temperature, T3/9 (OR)	Compressor effici- ency, Mg (percent)	Run
					Ave	rage	engi	e run	ing t	ime, a	0 hrs						
6786 S	911 902 899 898	7798 7556 7363 7066	7657 7427 7217 6916	7509 7275 7086 6795	1439 1244 1252	1616 1607 1594	1475 1476 1474	5.064 4.919 4.717	8014 7998 7983	.637 .635	37.65 37.79 37.30	37.50 37.38	37.48 37.87	53.56	2056 1940 1803 1674	78.0 78.7 78.2 76.6	1 2 5 4
4000 1	55A I	7651	7516	7357		rage		10 Fun			.5 hrs		TO 08	100 00	20000	97. 0	
6725 6523 6259	908 901 906 895 873	7497 7313 7075 6780	7358 7172 6926 6621	7815 7038 6801 6503	1448 1374 1258	1627 1608 1603 1585 1580	1489 1474 1464	5.048 4.895 4.755	8022 8046 7960 7998 8078	.641 .654 .637	37.25 36.93 37.32	36.22 36.72 36.84 37.31 38.39	37.11 37.22 37.67	51.65 52.62 53.01 53.62 54.25	1957 1825 1692	77.8 77.8 78.0 76.9 75.0	5 6 7 8 9
					Ave	rage	engi	tuni	ning t	ime, ]	6.1 hr	· .					
6608	911 897 891 875	7542 7366 6977 6537	7412 7232 6838 6377	7261 7090 6712 6264	1453 1284 1084	1610 1685 1668	1488 1478 1464 1466	5.018	7976 8022 7998 8022	.635 .639 .637 .639	35.98 36.58 36.82 37.28	35.58 36.11 36.57 37.47	35.50	50.85 51.51 52.51 53.36	1944	77.8 77.6 76.9 74.8	10 11 12 13
5445	904	7210	7075	6932	1476				7991		35.81		35.46	50.52	1954	76.5	14
6118	894 681	6934 5477	6789 6324	6658 6208	1505 1127	1586 1561	1467	4.641	7998 8006	.637	36.36		36.38	51.57 52.12	1738	75.7 74.0	15 16
<u></u>			<u> </u>			rage		e runı			.5 hrs			,			,_]
5052 4911 4679	837 829 825 819 808	5725 5613 5480 5260 4871	5619 5509 5375 5152 4758	5505 5404 5272 5059 4575	1404 1314 1221	1114 1104 1094 1079 1066	987 985 982 976	5.630 5.502 5.266 4,895	8590 8606 8598 8598 8590	.685 .685 .684	28.27 28.43 28.41 27.80 29.00	27.76 28.00 27.95 28.21	28.01 28.24 28.19	55.29 55.85 55.83 56.36	2203 2069	75.0 74.9 74.9 73.7 72.0	17 18 19 20 21
						rage				ime,	.5 hrs			<del></del>	<del></del>		
3364 5252 3076	835 628 825 817 808	5765 5729 5626 5457 5196	3705 3563 3566 3392 3125	3615 3585 3486 3324 3063	1510 1444 1582 1256 1071	716 719 709 700 692	637 643 635 633 635 engir	5.855 5.765 5.657 5.395 4.947	8687 8679 8687 8695 8670	.691 .692 .692 .690	18.06 18.18 18.27 18.19 18.31	18.13 18.02 18.09 18.21	18.50 18.17 18.24	54.66 55.12 55.28 55.44 55.51	2299 2216 2025	73.4 73.8 73.1 71.7 69.9	22 23 24 25 26
8060 8	839	2282	2247	2196	1515	442	395	5.705	8679	.691	10.89		10.94	53.39	2400	71.3	27
2043 8 2000 8 1904 8	834 831 826 818	2274 2224 2144 2059	2235 2188 2105 2014	2184 2138 2060 1975	1487 1419 1510 1166	440 455 455	393 391 395 400	5.728 5.602 5.387 5.059	8719 8705 8687 8695	.594 .693 .692	10.98	10.90 10.89 10.95	11.00 10.99 11.03	53.84 53.89	2377 2276 2095	71.0 70.8 70.5 68.2	26 29 30 31



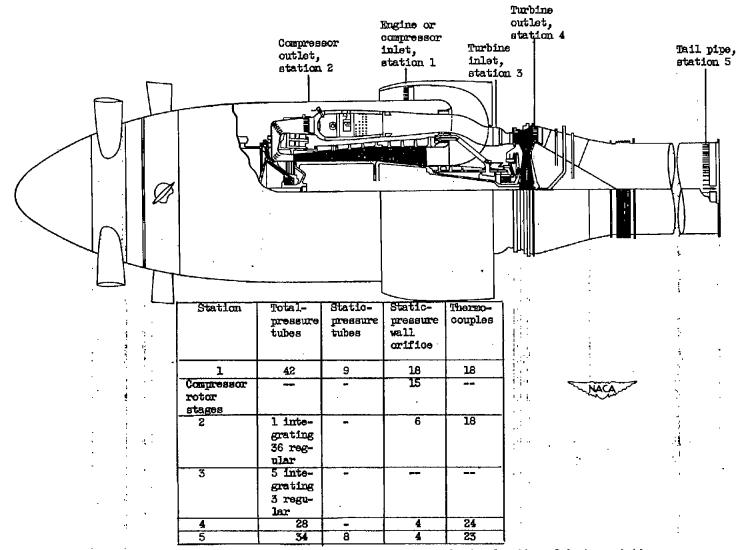


Figure 1. - Cross section of Python turbine-propeller engine showing location of instrumentation.

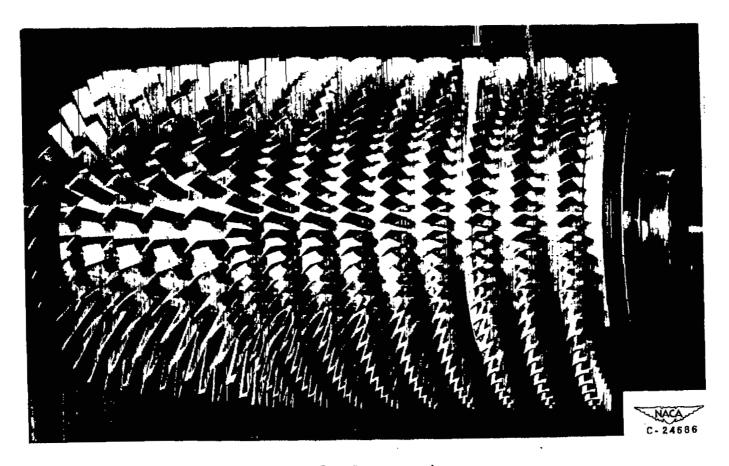


Figure 2. - Compressor rotor.

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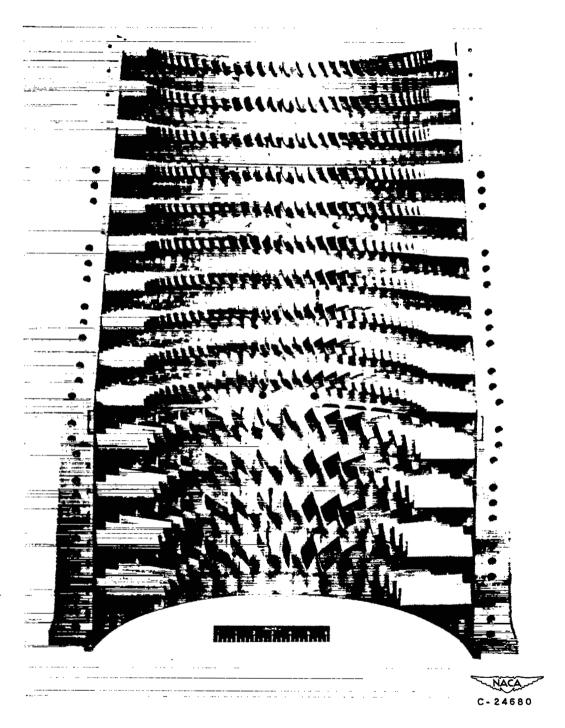


Figure 3. - Compressor stator (lower half).

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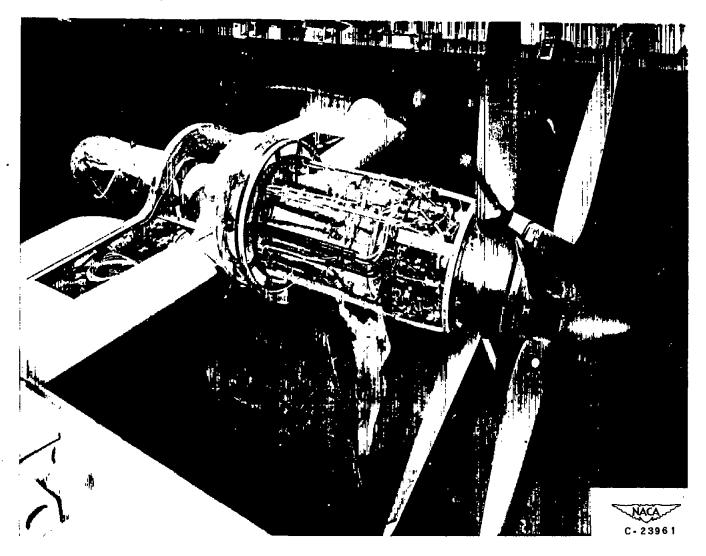
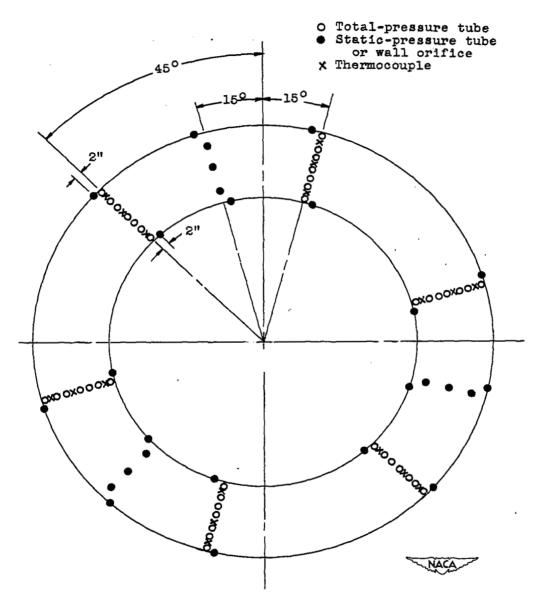


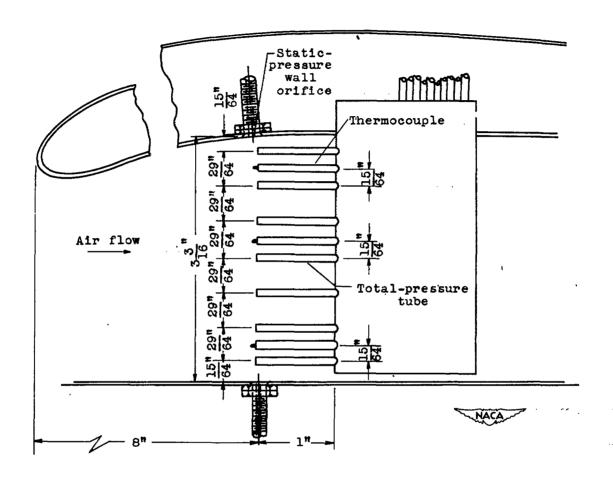
Figure 4. - Gas-turbine-propeller-engine installation in altitude wind turnel.

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(a) Schematic diagram; viewed from upstream.

Figure 5. - Instrumentation at engine inlet, station 1, 8 inches behind tip of cowling.

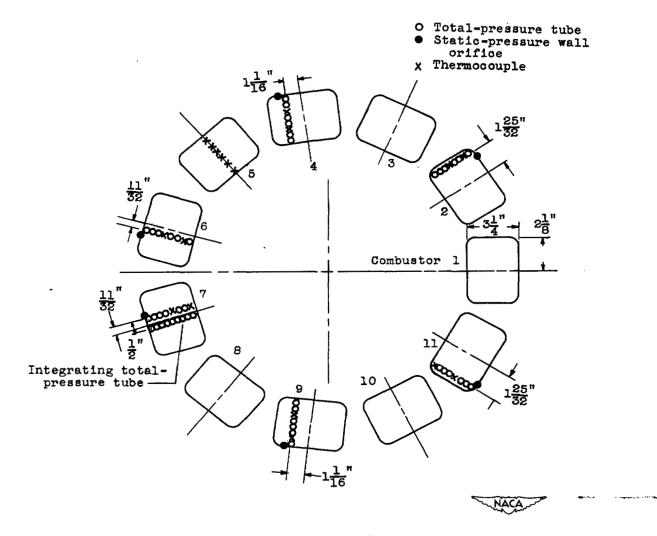


(b) Typical detail sketch of total-pressure tubes, static-pressure wall orifices, and thermocouples.

Figure 5. - Continued. Instrumentation at engine inlet, station 1, 8 inches behind tip of cowling.

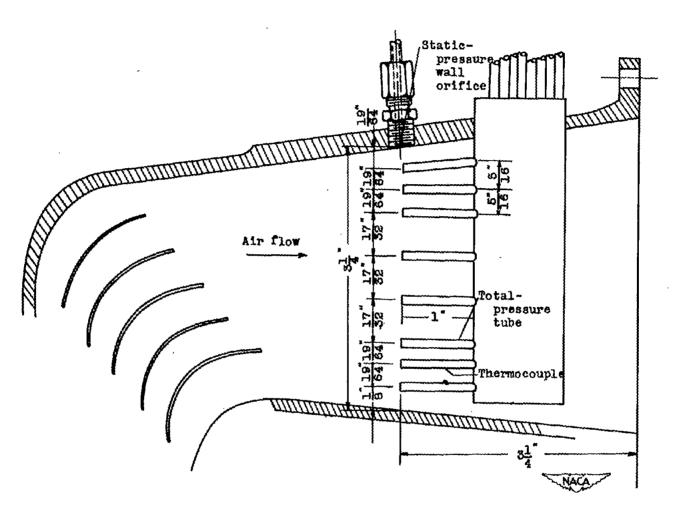
(c) Typical detail sketch of static-pressure tubes and wall orifices.

Figure 5. - Concluded. Instrumentation at engine inlet, station 1, 8 inches behind tip of cowling.



(a) Schematic diagram; viewed from upstream.

Figure 6. - Instrumentation at compressor outlet, station 2,  $3\frac{1}{4}$  inches upstream of burner-inlet flange.



(b) Typical detail sketch of total-pressure tubes, static-pressure wall orifices, and thermocouples.

Figure 6. - Concluded. Instrumentation at compressor-outlet, station 2,  $3\frac{1}{4}$  inches upstream from burner-inlet flange.

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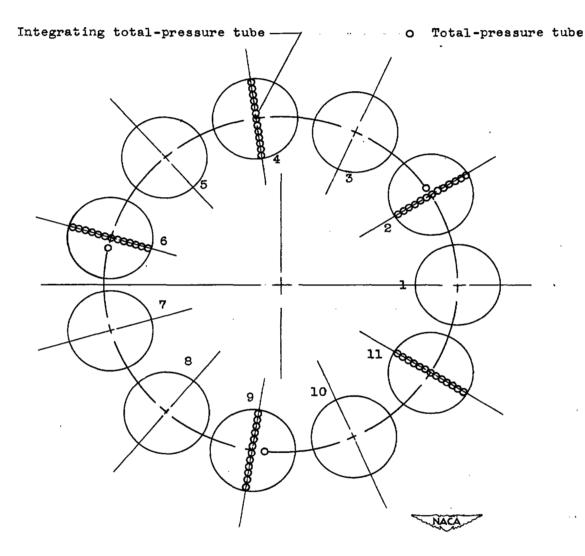


Figure 7. - Location of instrumentation at turbine inlet, station 3, 3 inches upstream of turbine flange. Viewed from upstream.

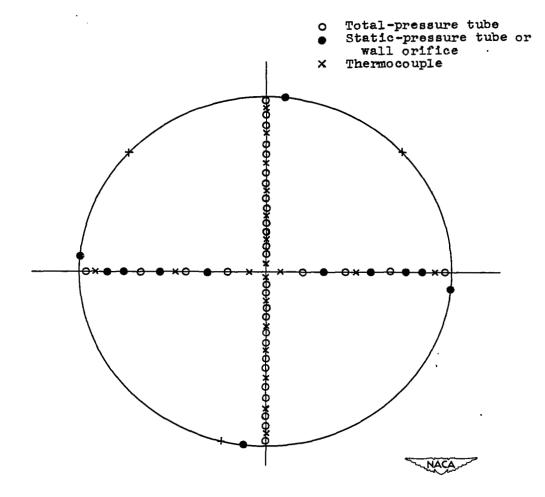


Figure 8. - Location of instrumentation in tail pipe, station 5,  $5\frac{1}{4}$  inches upstream of exhaust-mozzle outlet. Viewed from upstream.

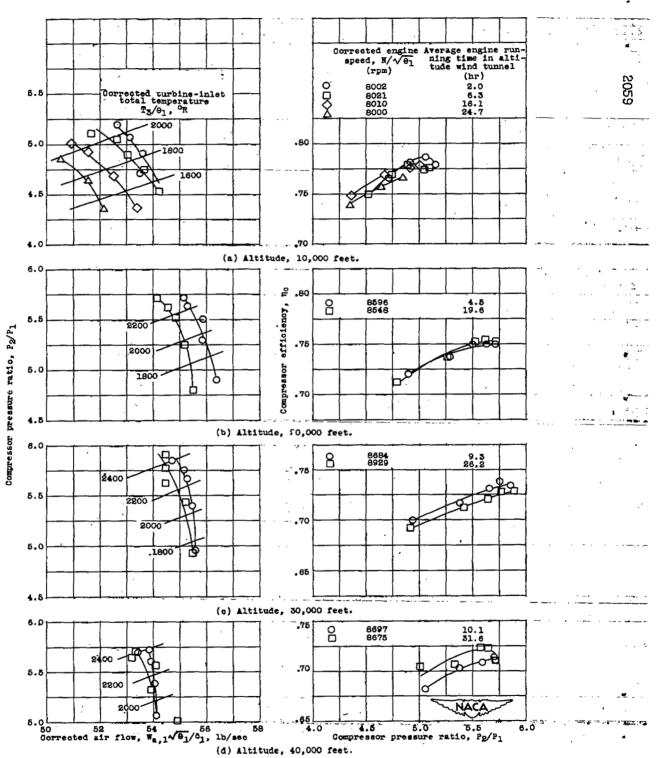


Figure 9. - Variation of compressor performance with engine operational time in altitude wind tunnel. Cowl-inlet ram pressure, 1.03.

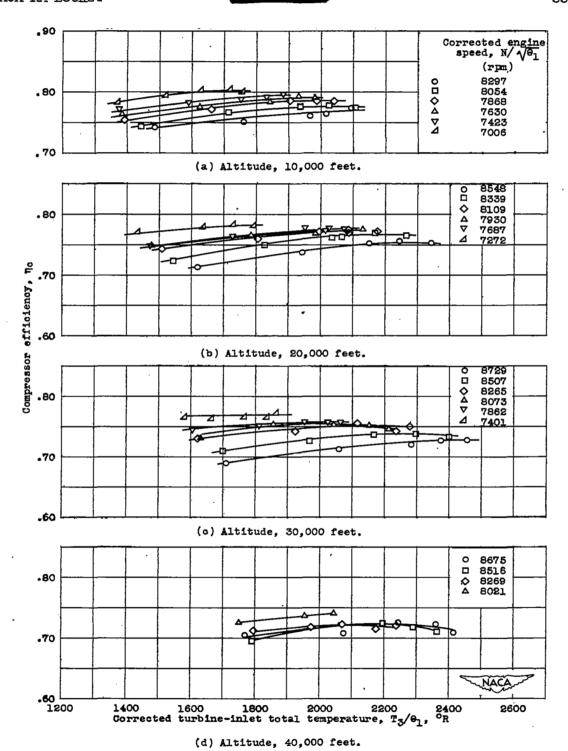
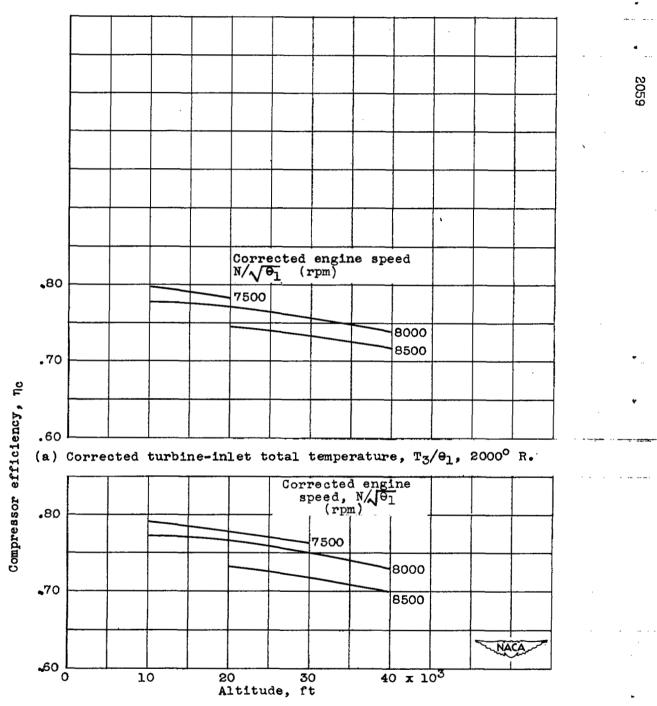


Figure 10. - Variation of compressor efficiency with corrected turbine-inlet temperature. Cowl-inlet ram pressure ratio, 1.03.



(b) Corrected turbine-inlet total temperature,  $T_3/\theta_1$ , 1800° R.

Figure 11. - Variation of compressor efficiency with altitude. Cowl-inlet ram pressure ratio, 1.03.

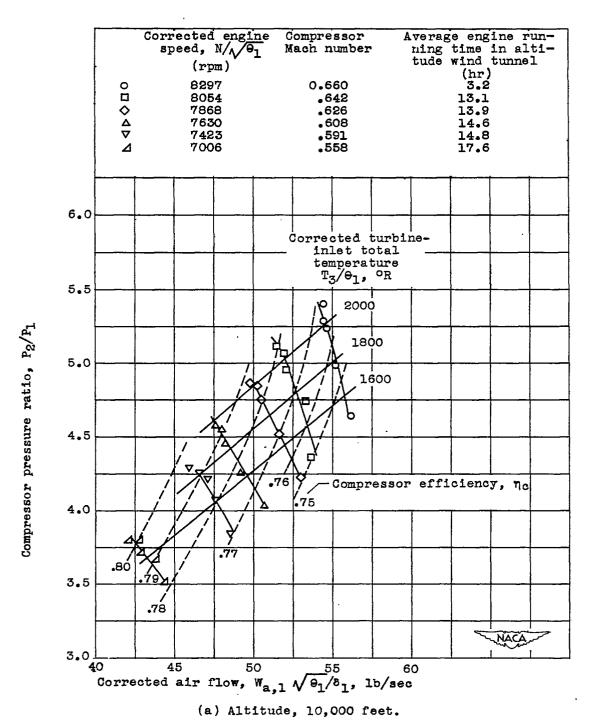
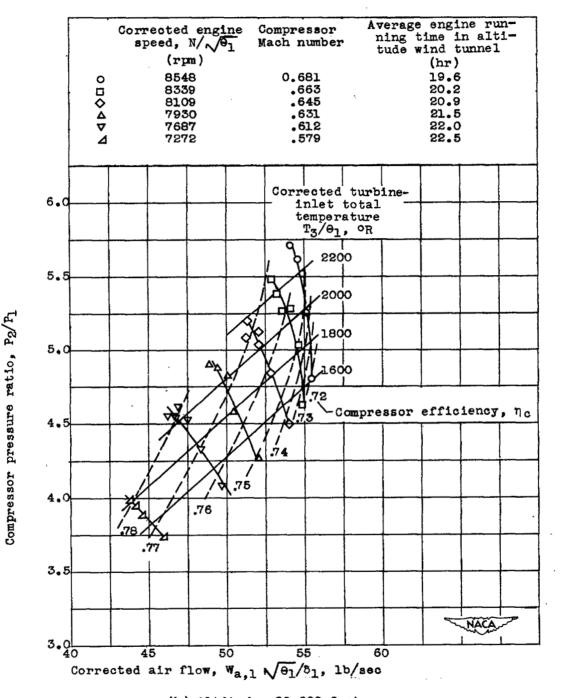


Figure 12. - Compressor performance map. Cowl-inlet ram pressure ratio, 1.03.



(b) Altitude, 20,000 feet.

Figure 12. - Continued. Compressor performance map. Cowl-inlet ram pressure ratio, 1.03.

Figure 12. - Continued. Compressor performance map. Cowl-inlet ram pressure ratio, 1.03.

(c) Altitude, 30,000 feet.

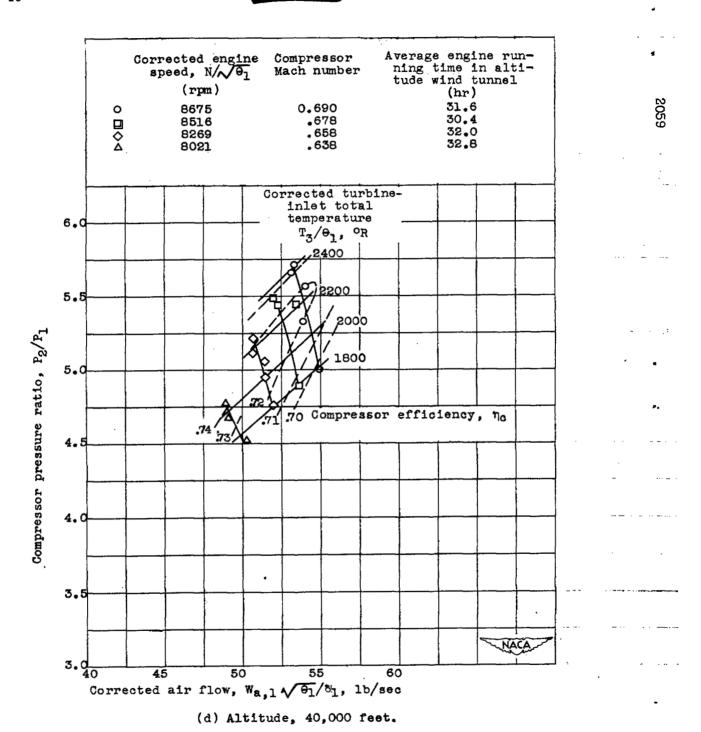
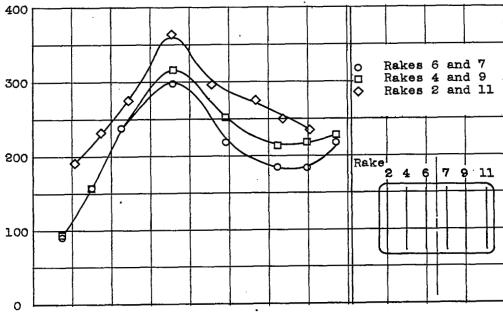
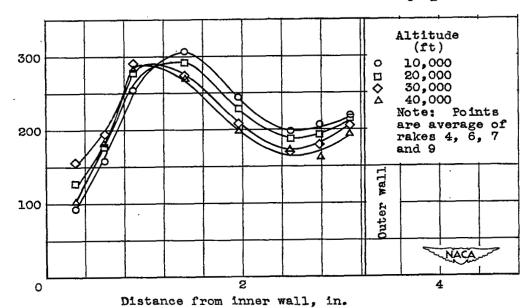


Figure 12. - Concluded. Compressor performance map. Cowlinet ram pressure ratio, 1.03.

Compressor-outlet velocity, V2, ft/sec

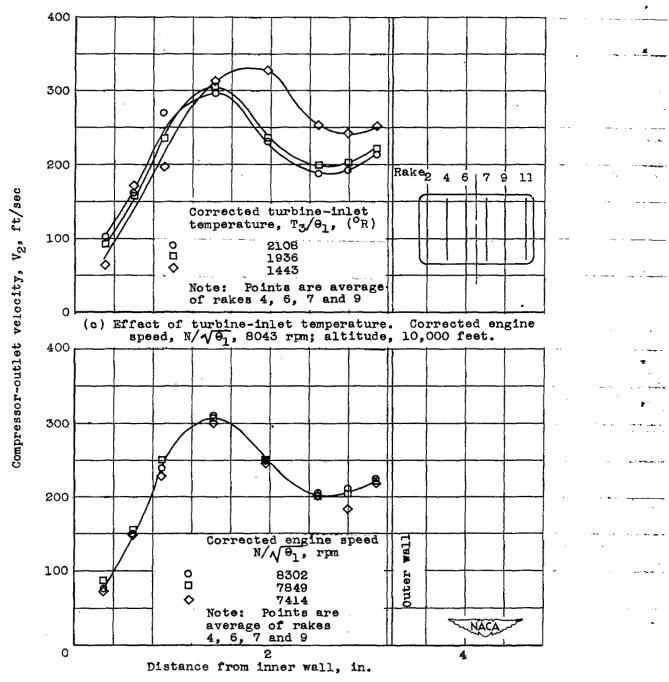


(a) Typical velocity profile at compressor outlet. Altitude 10,000 feet; corrected engine speed, N/ $\sqrt{\theta_1}$ , 8056 rpm; corrected turbine-inlet total temperature, T<sub>3</sub>/ $\theta_1$ , 1936 R.



(b) Effect of altitude. Corrected engine speed,  $N/\sqrt{\theta_1}$ , 8282 rpm; corrected turbine-inlet total temperature  $T_3/\theta_1$ , 2035 R.

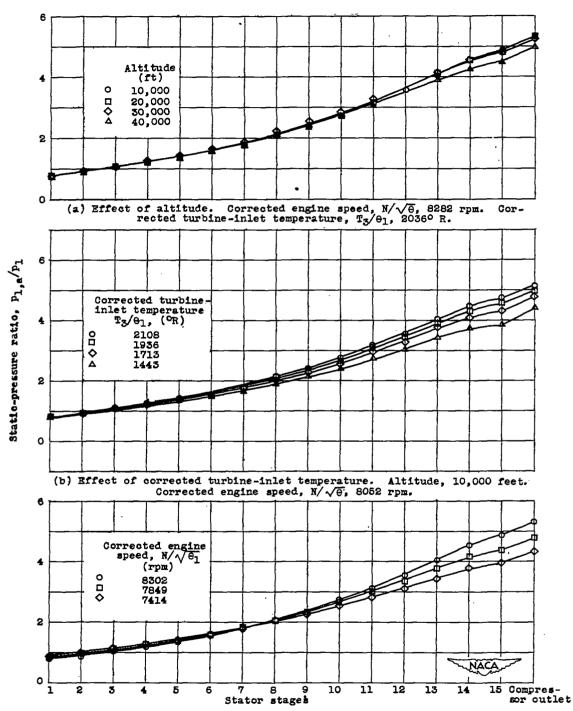
Figure 13. - Velocity profiles at compressor-outlet. Cowl-inlet ram pressure ratio, 1.03.



(d) Effect of corrected engine speed. Altitude, 10,000 feet; corrected turbine-inlet temperature,  $T_3/\theta_1$ , 1916 R.

Figure 13. - Concluded. Velocity profiles at compressor outlet.

Cowl-inlet ram pressure ratio, 1.03.



(c) Effect of engine speed. Altitude, 10,000 feet. Corrected turbine-inlet temperature (T3/01), 1916° R.

Figure 14. - Compressor-rotor stage static-pressure ratio profile. Cowl-inlet ram pressure ratio, 1.03.

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